

2506/305
AIRCRAFT MECHANICAL
TECHNOLOGY II
Oct. / Nov. 2022
Time: 3 hours



THE KENYA NATIONAL EXAMINATIONS COUNCIL
DIPLOMA IN AERONAUTICAL ENGINEERING
(AIRFRAME AND ENGINES OPTION)

MODULE III

AIRCRAFT MECHANICAL TECHNOLOGY II

3 hours

INSTRUCTIONS TO CANDIDATES

You should have the following for this examination:

Answer booklet;

Mathematical tables/Non-programmable scientific calculator;

Thermodynamic and transport properties of fluids tables, by G.F.C. Rogers and Y.R. Mayhew;

Drawing instruments.

This paper consists of EIGHT questions in TWO sections; A and B.

Answer THREE questions from section A and TWO questions from section B.

All questions carry equal marks.

Maximum marks for each part of a question are as indicated.

Candidates should answer the questions in English.

This paper consists of 5 printed pages.

Candidates should check the question paper to ascertain that all the pages are printed as indicated and that no questions are missing.

SECTION A: THERMODYNAMICS

Answer any **THREE** questions from this section.

1. (a) (i) With the aid of a diagram, describe the operation of an axial flow compressor.
- (ii) Show that the volumetric efficiency η_{vol} of a reciprocating air compressor is

$$\text{given by: } \eta_{vol} = 1 - \frac{V_c}{V_s} \left\{ \left(\frac{P_2}{P_1} \right)^{\frac{1}{n}} - 1 \right\}.$$

Where: V_c = clearance volume;
 V_s = swept volume;
 P_1 = suction pressure;
 P_2 = delivery pressure;
 n = index of compression and expansion.

(8 marks)

- (b) A two-stage single-acting reciprocating compressor compresses 1 kg/s of air from 1 bar and 18 °C to 15.5 bar. Compression is polytropic and $n = 1.3$ for both compression and expansion. The clearance volume is 4% of the swept volume in both stages and intercooling is perfect. If the intermediate pressure is ideal, determine the:

- (i) indicated power;
(ii) heat transferred through the low pressure cylinder walls;
(iii) cylinder total volumes required in the two stages.

(12 marks)

2. (a) (i) State the following gas laws:

- (I) Boyle's law;
(II) Charles' law.

- (ii) In a thermodynamic process, 0.2 kg of a gas at 140 kN/m² and 17 °C is compressed to a final pressure of 1.5 MN/m² according to the law $PV^{1.3} = \text{constant}$. The characteristic constant for the gas is 278 J/kgK and its adiabatic index is 1.4. Determine the:

- (I) initial and final volumes of the gas;
(II) final temperature;
(III) work transfer;
(IV) heat transfer.

(12 marks)

- (b) (i) Define the following terms:
- (I) wet vapour;
 - (II) dryness fraction;
 - (III) superheated steam.
- (ii) Steam with a dryness fraction of 0.95 and a pressure of 7 bar is supplied through a pipe of 25 mm diameter with a velocity of 12 m/s. Determine the mass flow rate of the steam. (8 marks)

3. (a) (i) Using a Temperature-Entropy (T-S) diagram, describe the Carnot cycle as applied to a heat engine.
- (ii) Show that the thermal efficiency of the Carnot cycle η_{carnot} is given by the equation:

$$\eta_{\text{carnot}} = 1 - \frac{T_3}{T_1}.$$

Where T_3 is the minimum temperature and T_1 is the maximum temperature. (5 marks)

- (b) In an ideal diesel cycle, the compression ratio is 15. The pressure and temperature at the beginning of the compression are 1 bar and 17 °C respectively. The heat transfer to the air is 1470 kJ/kg. Sketch the cycle on a Pressure -Volume (P-V) diagram and determine the:

- (i) pressure and temperature at the end of each process;
- (ii) cycle efficiency;
- (iii) mean effective pressure.

Take for air: $C_p = 1.005 \text{ kJ/kgK}$, $C_v = 0.718 \text{ kJ/kgK}$ and $\gamma = 1.4$. (15 marks)

4. (a) With the aid of sketches, distinguish between open and closed gas turbine plants. (4 marks)

- (b) A gas turbine unit takes in air at 17 °C and 1.01 bar and the pressure ratio is 8/1. The compressor is driven by the High Pressure (HP) turbine and the Low Pressure (LP) turbine drives a separate power shaft. The isentropic efficiencies of the compressor, HP and LP turbines are 0.8, 0.85 and 0.83 respectively. The maximum cycle temperature is 650 °C. For the compression process, take $C_p = 1.005 \text{ kJ/kgK}$ and $\gamma = 1.4$; For the combustion process and for expansion take $C_p = 1.15 \text{ kJ/kgK}$ and $\gamma = 1.333$ respectively. Neglect the mass of fuel.

- (i) Draw the plant and T-S diagrams.
- (ii) Determine the:

- (I) pressure and temperature of the gases entering the power turbine;
- (II) net power developed;
- (III) work ratio;
- (IV) thermal efficiency.

(16 marks)

5. (a) A flat composite plate is made of two layers of aluminium and steel of 50 mm and 20 mm thickness respectively. The thermal conductivities of aluminium and steel are 205 W/mK and 45 W/mK respectively. The hot surface of aluminium side is in contact with hot liquid at 200 °C, the heat transfer coefficient of liquid film being 14 W/m²K. The cold surface of steel is in contact with liquid at 25 °C, the heat transfer coefficient of liquid film being 29 W/m²K. Determine the:

- (i) overall thermal resistance of the composite;
- (ii) heat transfer rate from hot liquid to cold liquid through a surface area of 10 m².
(4 marks)

- (b) The dry exhaust gas from an engine had the following composition by volume:

$$\text{CO}_2 = 8.85\%; \text{CO} = 1.2\%; \text{O}_2 = 6.8\%; \text{N}_2 = 83.15\%.$$

The fuel had a percentage composition by mass of:

$$\text{C} = 84\%; \text{H}_2 = 14\%; \text{O}_2 = 2\%.$$

Determine the mass of air supplied per kg of fuel burnt.

Take: air contains 23% O₂ by mass.

(16 marks)

SECTION B: FLUID MECHANICS

Answer any TWO questions from this section.

6. (a) Experiments have shown that the resistance force F to the motion of a body moving through a fluid is proportional to the mass density ρ , dynamic viscosity μ of the fluid, velocity of the fluid V past a body of characteristic length l and acceleration due to gravity g . By use of dimensional analysis, show that the force $F = \rho v^2 l^2 \phi \left\{ \frac{vl\rho}{\mu}, \frac{lg}{v^2} \right\}$;
where $\phi =$ "function of"
(14 marks)

- (b) An aircraft whose wing measures 12 m is required to fly at an average speed of 800 km/hr. Its drag force is to be estimated from tests in water on a third scale model. Determine the drag force on the full size aircraft, if the drag force on the model is 1500 N.

Take the:

- densities of air and water as 1.225 kg/m³ and 1000 kg/m³;
- dynamic viscosities of air and water as 1.8×10^{-5} and 1.31×10^{-3} respectively.

(6 marks)

7. (a) Define the following types of efficiencies in centrifugal pumps:

- (i) manometric;
- (ii) mechanical;
- (iii) overall.

(3 marks)

(b) Show that the specific speed N_s of a centrifugal pump is given by: $N_s = \frac{NQ^{\frac{1}{2}}}{H^{\frac{3}{4}}}$.

Where: Q = flow rate in the pump;
 H = head;
 N = rotational speed of the impeller.

(7 marks)

(c) A centrifugal pump running at 100 rev/min is working against a total head of 20 m. The external diameter of the impeller is 500 mm and outlet width 60 mm. If the blade angle at outlet is 40° and manometric efficiency is 75%, determine the:

- (i) velocity of flow at outlet;
- (ii) absolute velocity of water leaving the blade;
- (iii) angle made by the absolute velocity at outlet with the direction of motion;
- (iv) flow rate through the pump.

(10 marks)

8. (a) From first principles, show that the loss of pressure P due to laminar flow in a horizontal

pipe is given by: $P = \frac{32\eta VL}{d^2}$.

Where: η = coefficient of viscosity of the fluid;
 V = mean velocity of flow of the fluid;
 d = diameter of the pipe;
 L = length of the pipe.

(14 marks)

(b) Oil of dynamic viscosity 0.097 kg/ms and relative density 0.93 flows through a horizontal circular pipe of diameter 150 mm and length 12 m. If the oil flow rate of 3.4 kg/s is collected in the tank, calculate the differences in pressure at the two ends of the pipe. (6 marks)

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