

2506/305

**AIRCRAFT MECHANICAL
TECHNOLOGY II**

June/July 2019

Time: 3 hours



**THE KENYA NATIONAL EXAMINATIONS COUNCIL
DIPLOMA IN AERONAUTICAL ENGINEERING
(AIRFRAMES AND ENGINES OPTION)**

MODULE III

AIRCRAFT MECHANICAL TECHNOLOGY II

3 hours

INSTRUCTIONS TO CANDIDATES

You should have the following for this examination:

Answer booklet;

Mathematical tables/Non-programmable scientific calculator;

Thermodynamic and Transport properties of fluid tables by Rogers and Mayhew;

Drawing instruments.

This paper consists of EIGHT questions in TWO sections; A and B.

Answer FIVE questions choosing THREE questions from section A and TWO questions from section B.

All questions carry equal marks.

Maximum marks for each part of a question are as shown.

Candidates should answer the questions in English.

This paper consists of 5 printed pages.

Candidates should check the question paper to ascertain that all the pages are printed as indicated and that no questions are missing.

SECTION A: THERMODYNAMICS

Answer any **THREE** questions from this section.

1. (a) Outline the **three** classes of fuels and list **two** examples for each. (6 marks)
- (b) Define the following terms as used in combustion of fuels:
- (i) stoichiometric air fuel ratio;
 - (ii) mixture strength;
 - (iii) flash point. (3 marks)
- (c) A petrol engine uses ethylalcohol C_2H_6O as the fuel. If the mixture strength is observed as 80%, determine the:
- (i) stoichiometric air fuel ratio;
 - (ii) actual air fuel ratio;
 - (iii) wet analysis by volume of the exhaust gases. (11 marks)

2. (a) Distinguish between impulse and reaction turbines on the basis of their operation. (4 marks)
- (b) With the aid of a velocity diagram of an impulse turbine, show that the diagram efficiency of a turbine η_d is given by:

$$\eta_d = \frac{2\mu(CW_i + CW_e)}{C_{ai}^2}$$

Where: μ is the mean blade speed;

CW_i and CW_e are velocity of whirl at inlet and exit of the steam respectively;

C_{ai} is the absolute velocity at inlet. (9 marks)

- (c) The mean blade speed of a reaction turbine at a particular speed is 75 m/s. The steam enters the turbine at a pressure of 3 bar and a temperature of 150 °C. The moving blade and fixed blades at this stage have inlet and exit angles at 35° and 25° respectively. Calculate the:
- (i) blade height at this stage if the height is $\frac{1}{12}$ of mean blade ring diameter;
 - (ii) power developed by the turbine when steam flow rate is 13.8 kg/s;
 - (iii) diagram efficiency. (7 marks)

3. (a) Explain the principle of operation of the following heat exchangers:
- (i) regenerative;
 - (ii) recuperative. (4 marks)
- (b) Using a temperature variation along the length of a parallel heat exchanger, derive the expression for logarithmic mean temperature difference (L.M.T.D). (9 marks)

- (c) A parallel flow heat exchanger carries hot and cold water streams at a rate of 0.25 and 0.6 kg/s respectively. The inlet temperature of the hot and cold streams are 80 °C and 30 °C respectively, while the exit temperature of the hot stream is 45 °C. If the heat transfer coefficient on both sides is 680 W/m²K and the specific heat capacity for the water is 4.2 kJ/kgK. Determine the pipe surface area required.

(7 marks)

4. (a) With the aid of a P.V diagram showing the liquid, wet steam and superheated steam regions, explain the process of steam generation in a boiler. (6 marks)

- (b) (i) Define the following terms as used in steam:

- (I) specific volume;
(II) dryness fraction.

- (ii) Show that the volume of wet steam V , can be expressed as:

$$V = V_g - (1-x)V_{fg}$$

Where: V_g is the volume of dry steam;
 V_{fg} is the volume of steam in the wet region.

(7 marks)

- (c) 1500 kg of steam at a pressure of 20 bar and dryness fraction of 0.85 is generated by a boiler in one hour. If the temperature of boiler feed water is 40 °C, determine the heat supplied per hour to generate wet steam. (7 marks)

5. (a) (i) State the following as used in perfect gases:

- (I) Joule's law;
(II) Avogadro's hypothesis.

- (ii) Distinguish between specific heat capacity at constant volume and specific heat capacity at constant pressure.

(4 marks)

- (b) Show that for a perfect gas, the specific heat capacity C_v is given by:

$$C_v = \frac{R}{\gamma - 1},$$

where: R is the characteristic gas constant;
 γ is the adiabatic index;
 C_v is the specific heat capacity at constant volume.

(8 marks)

(c) A perfect gas has a molecular mass of 24 kg/mole and the adiabatic index $\gamma = 1.2$. Calculate the heat loss under the following processes:

- (i) when the gas pressure decreases from 4.1 bar and 322 °C to 1.5 bar at a constant volume;
- (ii) when the gas enters the pipeline at 250 °C and exits at 25 °C at a constant pressure.

Neglect changes due to velocity of the gas.

Take: $R_o = 8314.4 \text{ J/kmolK}$

(8 marks)

SECTION B: FLUID MECHANICS

Answer any TWO questions from this section.

6. (a) Show that the mean velocity of viscous flow V between two parallel fixed plates due to a pressure P , in a dashpot with plates h distance apart and length L is given by:

$$V = \frac{Ph^2}{12\eta L},$$

where: η is the kinematic viscosity of the fluid.

(17 marks)

- (b) A dashpot consists of a piston 85 mm long sliding in a cylinder 50 mm diameter with a radial clearance of 1 mm. The cylinder is filled with a fluid of kinematic viscosity 0.2 kg/m-s. If the piston is acted upon by a force of 196.2 N, determine the velocity of the piston.

(3 marks)

7. (a) (i) List **three** applications of dimensional analysis.
 (i) A fluid of mass density ρ , coefficient of kinematic viscosity μ , flows through a circular pipe of diameter d and length L at a velocity V . Show by dimensional analysis that the pressure P in the pipe is given by:

$$P = \rho V^2 d \phi\left(\frac{VdL}{\mu}, \frac{L}{d}\right), \text{ where: } \phi \text{ is a function of.} \quad (13 \text{ marks})$$

- (b) A water pipe 40 m long and 20 mm diameter carries water flowing at a velocity of 5 m/s. The loss of head in the pipe is 25 m of water. Determine the loss of head in a 80 mm diameter pipe, 30 m long when carrying air.

Take: for air $\rho = 1.3 \text{ kg/m}^3$, $\mu = 1.8 \times 10^{-5} \text{ Ns/m}^2$;
 for water $\rho = 1000 \text{ kg/m}^3$, $\mu = 1.2 \times 10^{-3} \text{ Ns/m}^2$.

(7 marks)

8. (a) Define the following as applied in pumps:

- (i) manometric efficiency;
- (ii) mechanical efficiency;
- (iii) overall efficiency.

at the pump due to the impeller swept volume
 v_{t1} v_{t2}

(3 marks)

(b) A centrifugal pump has internal and external radii of the impeller r_1 and r_2 respectively. The corresponding peripheral velocities are μ_1 and μ_2 respectively. Derive the expression for the work done per unit weight of the fluid in terms of outer peripheral speeds μ_2 and the velocity of whirl W_2 .

(8 marks)

(c) The outer diameter and width of vanes set backwards at 75° of a centrifugal blower are 520 mm and 85 mm respectively. When the speed is 980 rev/min, a pressure difference of 40 mm of water is observed and the delivery of air is $3.5 \text{ m}^3/\text{s}$. The power supplied to the blower shaft is 1.7 kW.

Neglecting the vane thickness at inlet and assuming radial inlet, determine the:

- (i) manometric efficiency;
- (ii) power efficiency;
- (iii) overall efficiency.

Handwritten notes:
 $\eta = \frac{H_m \times g}{v_{t2} \times v_{w}}$
 $k1 = \frac{v_{t2} \times v_w}{g}$
 $P_w = k1 \phi H_m$
 $\tan \phi = \frac{x}{85}$
 $\tan 75 = \frac{40}{85}$
 0.1040

(9 marks)

THIS IS THE LAST PRINTED PAGE.

Handwritten notes:
 $v_{t2} = \frac{\pi D_2 N}{60}$
 $\frac{v_{t2}}{D_2} = \frac{v_{t1}}{D_1}$

Handwritten notes:
 $v_{t2} = \frac{\pi \times 0.52 \times 1000}{60}$
 $\frac{\pi \times 0.52 \times 1000}{60} \times 980$